## 3 series HLPSL &HLPSV LOAD SENSING PROPORTIONAL MULTI-WAY VALVE



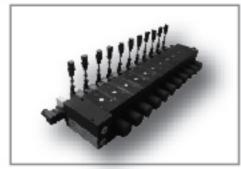




Friction positioning

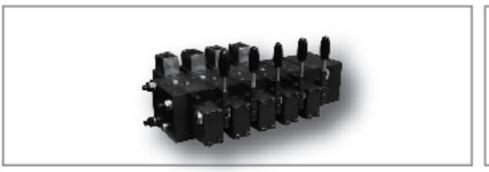


hydraulic control+Manual

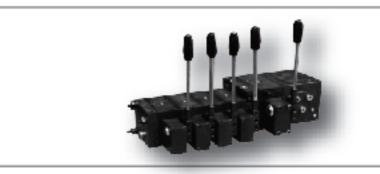


Manual control+Spring restoration

# 5 series HLPSL &HLPSV LOAD SENSING PROPORTIONAL MULTI-WAY VALVE



Manual+Electric proportional



5 series Via 3 series

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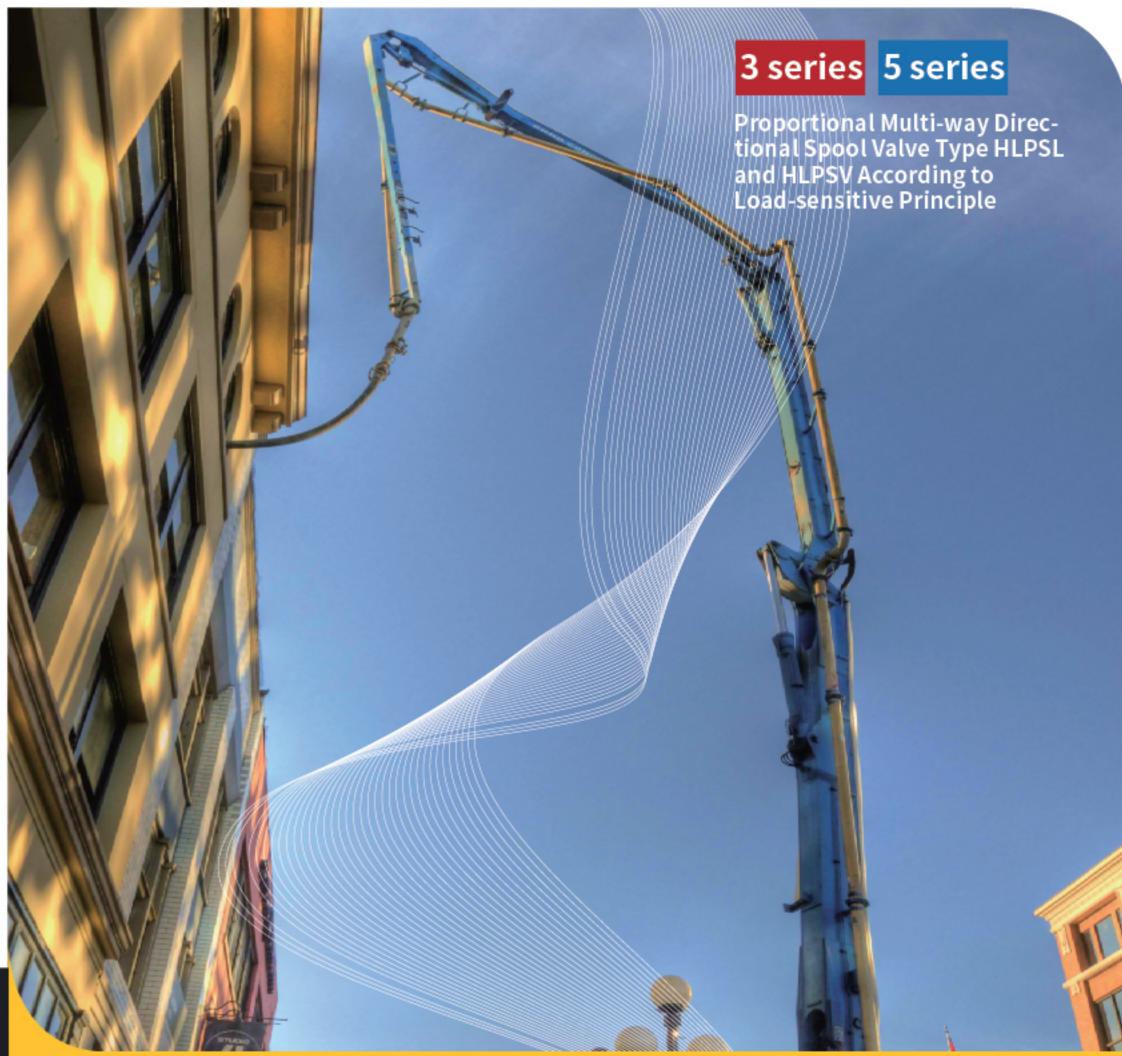
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# KM HYDRAULIC













## Table of Contents

For Snow Removing Machinery
STOP
5 series HLPSL &HLPSV LOAD
5 series HLPSL &HLPSV LOAD SENSING PROPORTIONAL MULTI-WAY VALVE

3 series	Proportional Multi-way Directional Spool Valve Type HLPSL and HLPSV According to Load-sensitive Principle	1
I.Overview		1
II. Model S	election Code ·····	3
	ral Style ·····	
	arameters	
	Dimension · · · · · · · · · · · · · · · · · · ·	
	tion Example · · · · · · · · · · · · · · · · · · ·	_
	Characteristics	
VIII. Explos	sive Drawing ······	38
IX. Matters	Needing Attention for Use ·····	39
X. Treatme	nt of Simple Faults	39
5 series	Proportional Multi-way Directional Spool Valve Type HLPSL and HLPSV According to Load-sensitive Principle	11
	Proportional Multi-way Directional Spool Valve Type HLPSL and HLPSV According to Load-sensitive Principle	
I.Overview	HLPSV According to Load-sensitive Principle	41
I.Overview II. Model S	HLPSV According to Load-sensitive Principle	41 43
I.Overview II. Model S III. Structu	HLPSV According to Load-sensitive Principle	41 43 47
I.Overview II. Model S III. Structu IV. Other P	HLPSV According to Load-sensitive Principle  election Code  ral Style	41 43 47 57
I.Overview II. Model S III. Structu IV. Other P V.Outline I	HLPSV According to Load-sensitive Principle election Code ral Style	41 43 47 57 63
I.Overview II. Model S III. Structu IV. Other P V.Outline I VI. Applica VII. Other (	HLPSV According to Load-sensitive Principle  election Code ral Style arameters bimension tion Example Characteristics	41 43 47 57 63 71 74
I.Overview II. Model S III. Structu IV. Other P V.Outline E VI. Applica VII. Other ( VIII. Explos	HLPSV According to Load-sensitive Principle  election Code ral Style arameters Dimension tion Example Characteristics	41 43 47 57 63 71 74 75
I.Overview II. Model S III. Structu IV. Other P V.Outline I VI. Applica VII. Other ( VIII. Explosi	HLPSV According to Load-sensitive Principle  election Code ral Style arameters bimension tion Example Characteristics	41 43 47 57 63 71 74 75 76

Jiangsu KM Hydraulic Control Sys. Co.,Ltd



### Proportional Multi-way Directional Spool Valve Type HLPSL and HLPSV According to Load-sensitive Principle

Pmax=420bar(6000psi)

Qmax=210 L/min(55g/min)

Combined Type Size 5

#### I. Overview

Proportional multi-way directional spool valve type HLPSL and HLPSV according to load-sensitive principle are mainly applied to engineering machinery electric hydraulic control field to control the movement direction & speed of hydraulic oil cylinder or hydraulic motor. With various functions such as electric proportional control, electric switching value control, hydraulic control, pneumatic control, manual control, etc., such valves can realize stepless speed regulation and will not be influenced by load change. The characteristic of load sensing enables the multi-way actuators to run simultaneously and independently from each other at different speed and pressure ratings until the maximum flow of bumping source is reached.

Load-sensitive principle, applied in hydraulic system, can realize flow control of actuators without any influence of the load. Load-sensitive mechanism (fixed differential reducing valve) enables the outlet pressure to match with the changing load continuously during the working period. One side is with the function of load signal & spring force, while the other side is imposed on outlet pressure of fixed differential reducing valve which is equal to the sum of load pressure and the pressure generated by spring, namely, the differential pressure beside the throttle of proportional valve is a constant value and therefore the flow of proportional valve is only in proportional to control signal. As to the valve type HLPSL, the load pressure signal passes shuttle valve, and the damper exerts a function on the spring cavity of fixed differential overflow valve which will regulate the system pressure to the value of the sum of load pressure and spring force, so that the outlet pressure of pump can match with the load pressure. In case the load pressure is low, the outlet pressure of pump will be low and vice verse, so as to realize the energy conservation of the system.

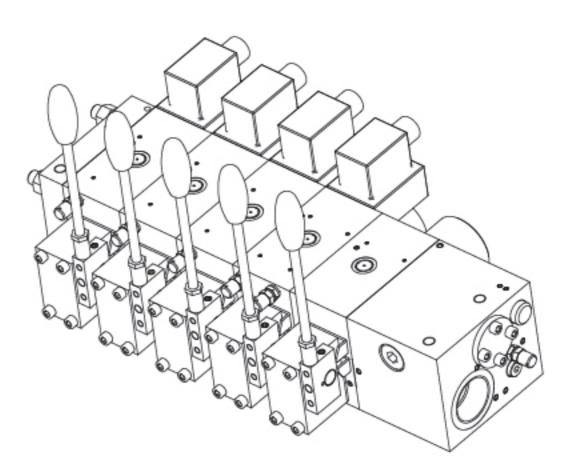
As to multi-way directional spool valve type HLPSV, the directional spool valve of each way, after selection of load pressure by shuttle valve, will deliver the maximum load pressure signal to load-sensitive port X of the variable displacement pump, and make the output pressure & flow of variable displacement pump match with the system through the control of pump's variable displacement mechanism, so as to realize the energy conservation of the

Compared to the valve type HLPSL, valve type HLPSV is slightly slower in the response, which is because the pump's variable displacement mechanism shall make regulation with action hysteresis. However, valve type HLPSV has better effect of energy conservation.



Valve type HLPSL is applied to constant displacement pump system, while valve type HLPSV shall be used for variable displacement pump system.





Proportional multi-way directional spool valve according to load-sensitive principle is a kind of combined-type valve, which generally consists of 3 functional elements, namely, connection block, directional spool valve and end plate. If necessary, ancillary block can be mounted on the directional spool valve additionally and intermediate block can be added between directional spool valves.

#### Connection block

Oil-taking way of multi-way valve is equipped with pressure oil inlet P and oil return tank port R, as well as control port LS and measurement ports Z & M. (Please refer to Section 3.1 for detailed information.)

#### Model selection

- n compliance with the variety of oil source, constant displacement pump and variable displacement pump
- In compliance with the specifications of connecting threads of port P and port R, G1, G1 1/4
- With or without pilot controlled oil supply
- With or without pressure limitation
- · With or without electromagnetic unloading of the pump

— Combined Type Size 5 —— — Combined Type Size 5 ———



#### Directional spool valve

Proportional directional valve shall be fixed between connection block and end plate by three screw stems (pull rods) and 12 directional spool valves can be mounted for a valve bank at most. Generally, the number of ways of the directional spool valve doesn't exceed 8. In case the number is equal to or larger than 9, it's suggested to divide the valves into two valve banks of multi-way valve. As to the directional spool valve with 9 or more ways, in case the system can't be divided into two valve banks of multi-way valve, it's required to add ZPL33/15 intermediate block or order corresponding enhanced-type pull rod accessory. (Please refer to Section 3.2 for detailed information.)

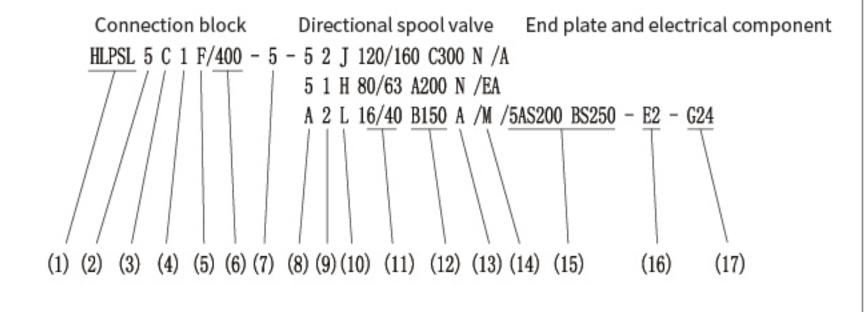
- In accordance with the neutral-position function and flow of directional spool valve
- With or without secondary pressure limitation, N no pressure limitation, C-type pressure limitation and AB-type pressure limitation
- In accordance with the operation manner, pure manual control, electronic control, hydraulic control & friction positioning
- With or without any auxiliary function

#### END PLATE

As the terminal of combined-type multi-way valve, the end plate shall be selected according to the characteristics as follows: with oil return port T controlling the internal discharge or external discharge of oil, with or without additional LS, oil inlet P, oil return port R, etc. (Please refer to Section 3.3 for detailed information.)

- With port T for pilot control of internal discharge or external discharge of return oil
- With or without additional LS inlet or cut-off of pump's circulation loop

#### II. Model Selection Code





#### Basic code of connection block

HLPSL used for oil supply system of constant displacement pump (open circuit);

HLPSV used for oil supply system of variable displacement pump with flow regulator (closed circuit), or used in the same constant displacement pump system as a separate second multi-way valve bank

#### Threaded coupling of port P & port R of the connection block (in compliance with ISO288/1 standard) (please refer to Table 1 in 3.1 for details)

- G1
- G1 1/4

#### Added component (please refer to Table 2 in 3.1 for details)

- C standard type
- one-way throttle valve (only used for HLPSL)
- damping hole added in LS-oil way (only used for standard-type HLPSV and HLPSL)
- with throttling orifice in LS-oil way (only used for HLPSV)
- one-way throttle valve + unloading valve (only used for HLPSL)
- fixed differential overflow valve for improving the circulating pressure (about 14bar, HLPSL)
- pressure oil outlet with excessive flow of fixed differential overflow valve (only used for HLPSL)

#### Control oil supply pressure (please refer to Table 3 in 3.1 for details)

- in case of no three-way reducing valve and directional spool valve of pure manual operation, or when pilot control oil is provided externally, the pilot control pressure shall be 20bar-40bar.
- with three-way reducing valve, it's used for the supply of pilot control oil (control pressure shall be about 20bar).

#### Two-position two-way solenoid valve for system unloading (please refer to Table 4 in 3.1 for details)

- without solenoid valve
- normally open solenoid valve, in case of power supply, the pump is pressurized; in case of power failure, the pump carries out unloading
- normally closed solenoid valve, in case of power supply, the pump carries out unloading; in case of power failure, the pump is pressurized
- D. ./F.. with pressure limiting valve, it can serve as secondary pressure (F50) with pressure value indicated

———— Combined Type Size 5 ———— 



(6) Pressure setting of pressure limiting valve in the connection block

/··· regulated value of the pressure of pressure limiting valve (bar), such as 63, 120, 210, 280, 315, 350, 400 bar

/NF without pressure limiting valve (only referring to type HLPSV)

(7) Specification

size 5

Thread size of control oil ports A & B of directional spool valve, conforming to ISO228/1 standard (please refer to Table 5 in 3.2 for details)

5 G 1

suitable for the installation of ancillary block or intermediate transition block (intermediate block)

Intermediate block ZPL 5S(V)/H Transition

ZPL 5S(V)/E connection

block, thickness 9mm ZPL 55/9

#### Base block of directional spool valve and fixed differential reducing valve (please refer to Table 6 in 3.2 for details)

- 1 without inlet fixed differential reducing valve, only for single-way successive operation or multi-way asynchronous operation
- (standard type) with inlet fixed differential reducing valve and load compensation function, multi-way synchronous operation can be realized
- with inlet & outlet fixed differential reducing valve, only suitable for function N 3-position 3-way valve
- with inlet fixed differential reducing valve which has enhanced spring, to obtain relatively large flow output
- with inlet fixed differential reducing valve which has enhanced spring, to obtain relatively large flow output

Function symbol of spool (please refer to Table 7 in 3.2 for elementary function code)

Selectable with several types: L, M, F, H, J, B, R, O, N, etc.

Output flow code of ports A & B (please refer to Table 8 in 3.2 for details)

Maximum output flow of ports A & B (generally select 16, 25, 40, 63, 80,120,160 (L/min), or select the maximum flow within the range of 16-160 L/min according to user's requirement)

Secondary pressure limitation (please refer to Table 9 in 3.2 for details)

without pressure limiting protection

limit the pressure of actuator ports A & B at the same time, unit (bar)

A······B······ limit the pressure of actuator ports A & B respectively, unit (bar)



#### Functional cut-off and pressure extraction (please refer to Table 10 in 3.2 for details)

- N without functional cut-off and pressure extraction
- X C-type pressure limitation, with pressure extraction port X, thread size of oil port is G1/8 respectively
- S AB-type pressure limitation, with pressure extraction ports U & W, thread size of oil port is G1/8 respectively
- S1 AB-type pressure limitation, with flange plate extraction ports U & W, thread size of oil port is G1/8 respectively
- F1 electrical cut-off on the side of actuator port A
- F2 electrical cut-off on the side of actuator port B

#### (14)Operation manner (please refer to Table 11 in 3.2 for details)

/A(1,2)	manual operation with long handle (1=without handle, 2=with short handle)
/E	electric proportional control
/EA(1,2)	electric proportional control and manual operation (1=without handle, 2=with short handle)
/D	electric switching value control
/DA(1,2)	electric switching value control and manual operation (1=without handle, 2=with short handle)
/M	manual friction positioning with long handle
/H	pure hydraulic control (two pilot oil ports are in the same direction with oil ports A and B)
/F	pure hydraulic control (two pilot oil ports are perpendicular to oil ports A and B and outward respectively
/HA (1,2)	hydraulic control and manual operation (two pilot oil ports are in the same direction with oil ports A and
	B; 1=without handle, 2=with short handle)
/FA (1,2)	hydraulic control and manual operation (two pilot oil ports are perpendicular to oil ports A and B and
	outward respectively; 1=without handle, 2=with short handle)
/ A (H)	manual control, or hydraulic control, be delivered out of the factory with manual control
/P	pneumatic control
/PA(1,2)	pneumatic control and mnual operation (1=without handle, 2=with short handle)
/P1A(1,2)	pneumatic control and manual operation, pneumatic control port is in reverse direction with oil port A B
	(1=without handle, 2=with short handle)

#### Ancillary block (please refer to Table 12 in 3.2 for details)

/5ASBS	pressure limiting safety valve is mounted at ports A & B, thread size of oil ports is G1, with limited
	pressure, unit: bar
/5AN BN	with ancillary block of oil supplementary valve at ports A & B
/5AN	with cushion valve at port A and ancillary block of oil supplementary valve at port B
/5BN	with cushion valve at port B and ancillary block of oil supplementary valve at port A
/5AL,/5BL	with ancillary block of balance valve at ports A & B, with the pressure indicated, unit: bar
/5DRH	with ancillary block of hydraulic control one-way valve at ports A & B
/5DRHA	with ancillary block of hydraulic control one-way valve at port A
/5DRHB	with ancillary block of hydraulic control one-way valve at port B

ancillary block without any additional function, thread sizes of oil ports shall be G1 respectively

Combined Type Size 5 Combined Type Size 5



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#### (16)End plate (please refer to Table 13 in 3.3 for details)

- E1 standard end plate, external discharge of pilot oil, connecting oil return control port T externally
- similar to E1, external discharge of pilot oil, port Y is attached additionally
- external discharge of pilot oil, a 2-position 3-way magnetic ball valve is mounted; in case of power supply, the system pressure rises to the pressure of safety valve
- similar to E1, internal discharge of pilot oil, port T can be sealed
- similar to E2, internal discharge of pilot oil, port T can be sealed
- similar to E3, internal discharge of pilot oil, port T can be sealed
- E18 external discharge of pilot oil, with additional port Y, pressure oil port P and oil return port R
- E19 internal discharge of pilot oil, but the oil return pressure shall be less than 15bar, with additional port Y, pressure oil port P and oil return port R
- E20 external discharge of pilot oil, but the oil return pressure shall be less than 15bar, with additional port Y, pressure oil port P and oil return port R

#### Control voltage and the length of anti-explosion type cable (please refer to Table 14 in 3.3 for details) (17)

G12 12VDC

24VDC G24

24VDC (anti-explosion type, length of guide line: 3m, 5m, 10m, etc.)

### III. Structural Style and Key Data for Model Section

#### 3.1 Connection block

For example: HLPSL5 C F/ 210-5-52L80/40B250A/E-E1-G24

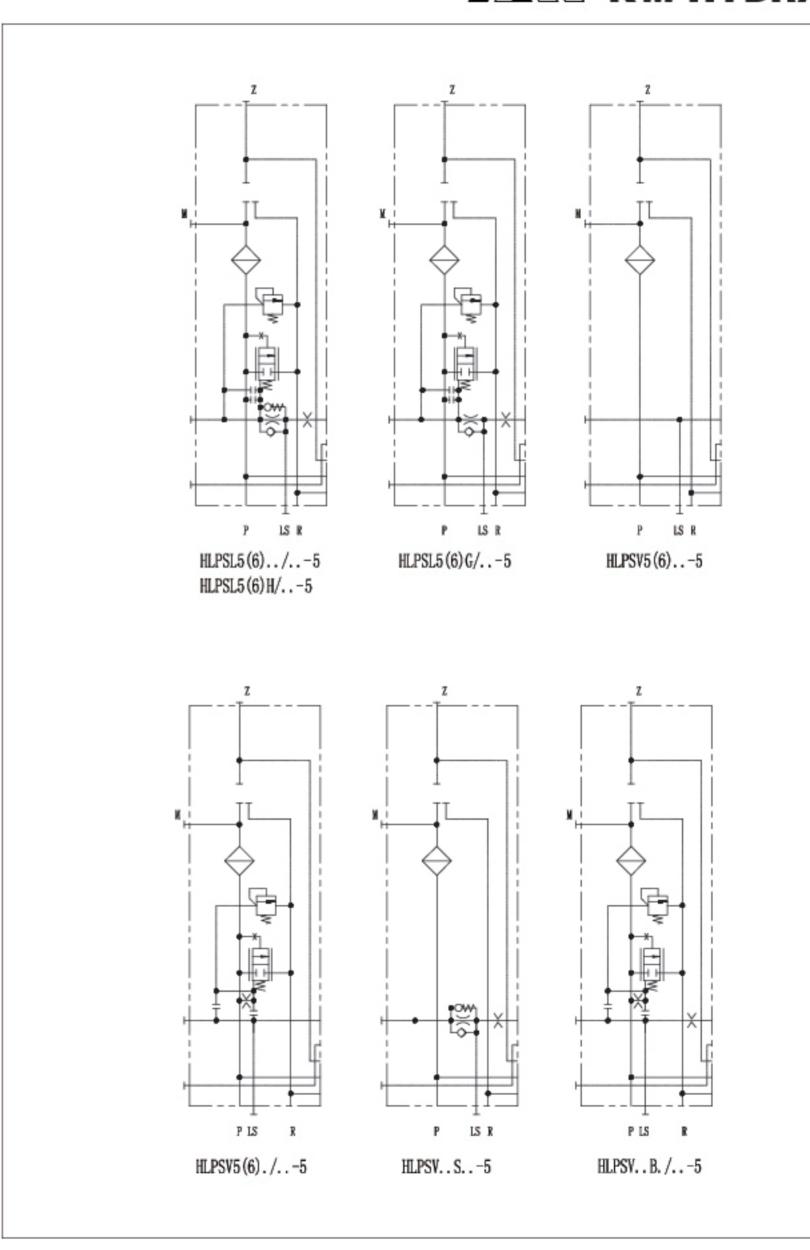
Table1 Table2 Table3 Table4 \*

\* Please refer to the statement in previous section "Model Selection Code (6)" for the pressure of pressure limiting valve

Table 1 Model of Connection Block and Thread Size of Oil Inlet & Oil Return Port

Code	Thread ports P & R	Maximum flow of the pump
HLPSL5, HLPSV5	G 1	250L/min
HLPSL6, HLPSV6	G 1 1/4	300L/min

Note: in case HLPSL5(6) and HLPSV5(6) are combined with the 1st directional spool valve with ancillary block, ZPL55/9 type transition plate shall be mounted behind the connection block, and otherwise the connector can't be mounted at port R.



— Combined Type Size 5 — Combined Type Size 5

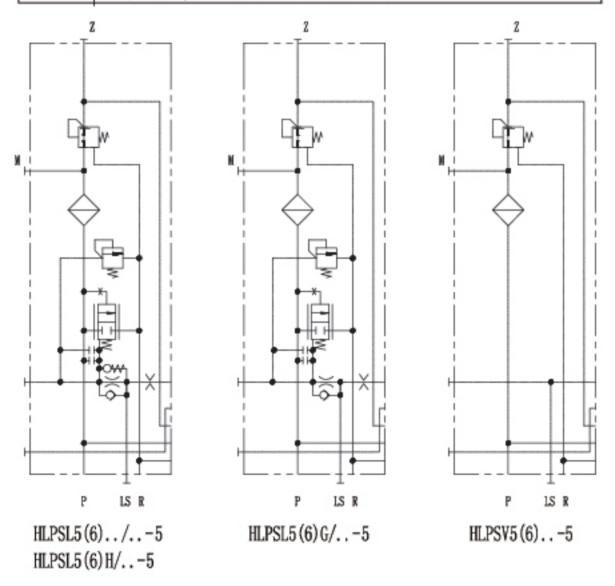


### Table 2 Code of Added Components of the Connection Block

Code	Description			
С	Standard type			
S Add damping hole in LS-oil way (only used for standard-type HLPSL and HI				
В	With 0.8mm throttling orifice in LS-oil way (only referring to type HLPSV)			
G	One-way throttle valve (only HLPSL)			
Z	One-way throttle valve + unloading valve (HLPSL)			
Н	Fixed differential overflow valve for improving the circulating pressure (about 14 bar, HLPSL)			
Υ	Pressure oil outlet with excessive flow of fixed differential overflow valve			

### Table 3 Pilot Oil Supply Code

Code Description				
0	In case of no three-way reducing valve and directional spool valve of pure manual operation, or when control oil is provided externally, the pilot control pressure shall be 20bar-40bar.			
1	With three-way reducing valve, it's used for the supply of pilot control oil (control pressure shall be about 20bar).			
2	With three-way reducing valve, it's used for the supply of pilot control oil (control pressure shall be about 40bar).			





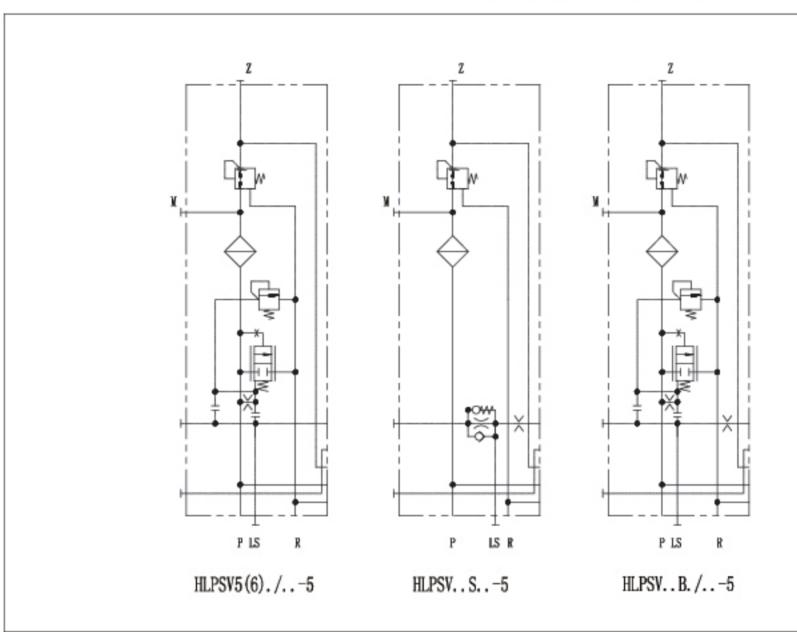
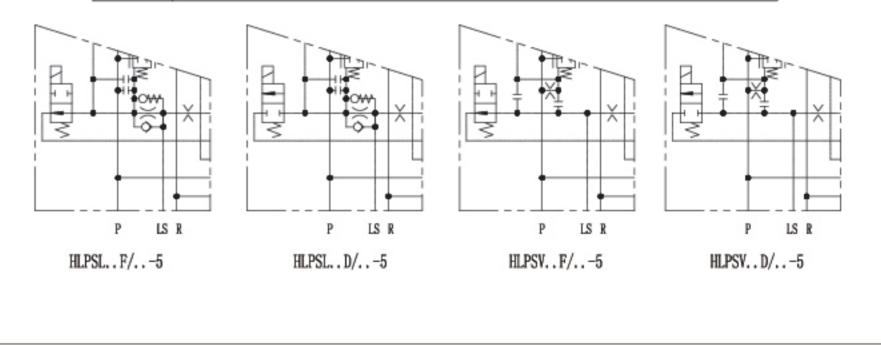


Table 4 Unloading solenoid valve and Pilot Pressure Limitation of the System

Description				
Without solenoid vale				
Normally open solenoid valve, in case of power supply, the pump is pressurized; in case of power failure, the pump carries out unloading (emergency stop)				
Normally closed solenoid valve, in case of power supply, the pump carries out unload- ing; in case of power failure, the pump is pressurized				
With pressure limiting valve, it can serve as secondary pressure (F50) with pressure value indicated, for example: as to type HLPSL41F100/350-3, in case of power failure, Pmax.=100bar; in case of power supply, Pmax.=350bar				



 Combined Type Size 5 Combined Type Size 5





### 3.2 Directional spool valve

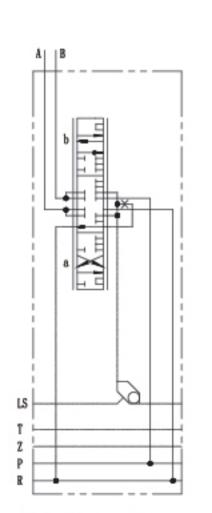
For example: HLPSL5 C 1 F/210–5 –  $\underline{5}$   $\underline{2}$  –  $\underline{L}$   $\underline{80/40}$   $\underline{B250}$   $\underline{A/}$   $\underline{EA1}$  /  $\underline{5DRHA}$  – E1– G24 Table5 Table6 Table7 Table8 Table9 Table10 Table11 Table12

### Table 5 Oil Port Size of the Directional Spool Valve

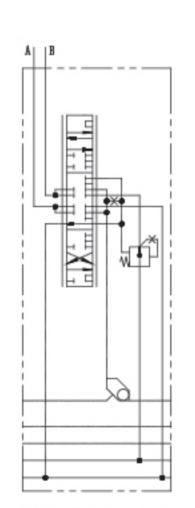
Thread code of oil port of directional spool valve	Thread ports A & B	Maximum flow and additional description
5	G	160L/min
A	Ancillary block port can be mounted	160L/min

### Table 6 Functions of Valve Block of Directional Spool Valve

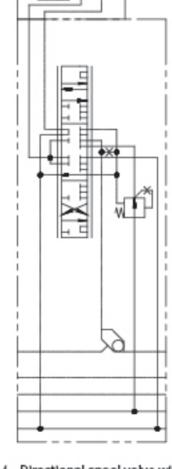
Code	Description
1	Without inlet fixed differential reducing valve, only for single-way successive opera- tion or multi-way asynchronous operation
2	(Standard type) with inlet fixed differential reducing valve and load compensation function, multi-way synchronous operation can be realized
4	With oil-taking & oil return fixed differential reducing valve mounted on the 3-posi- tion 3-way directional spool valve, it can realize the regulation of output flow & return oil flow without any influence of the load
5	With inlet fixed differential reducing valve which has enhanced spring, to obtain relatively large flow output
8	Pre-selector valve (output of port A; closed externally of port B to supply oil for subsequent directional spool valves; without fixed differential reducing valve)



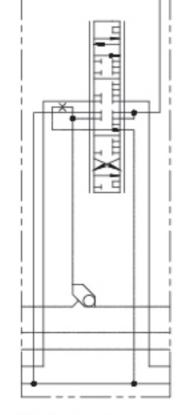
 Directional spool valve without inlet fixed differential reducing valve



2, 5. Directional spool valve without inlet fixed differential reducing valve



. Directional spool valve with oil-taking & oil return fixed differential reducing valve



Directional spool valve as pre-selection switch

### Table 7 Function Symbol of Spool

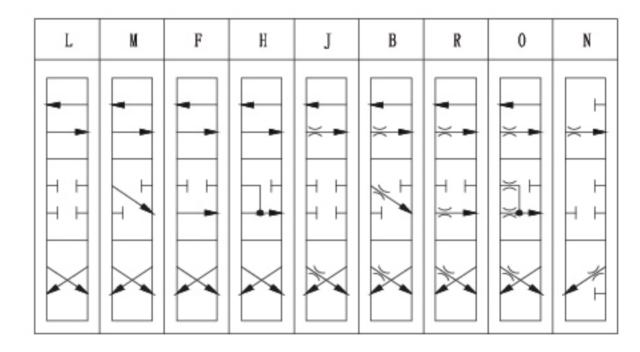


Table 8 Maximum Control Flow of Actuator P-A or P-B (L/min)

Base block of directional				Flow	code		
spool valve	16	25	40	63	80	120	160
1,5	20	32	51	80	110	150	210
2	16	25	40	63	80	120	160
4	16	25	40	63	80	120	160
8	Only refe	er to the f	low of po	rt A, identi	ical with t	ne value o	f 1, 5 in the table

Only refer to the flow of port A, identical with the value of 1, 5 in the table

N	Without pressure limiting protection
C	Limit the pressure of ports A & B, limited pressure value, unit (bar)
AB	Limit the pressure of ports A & B, limited pressure value, unit (bar)

— Combined Type Size 5 — — Combined Type Size 5 —



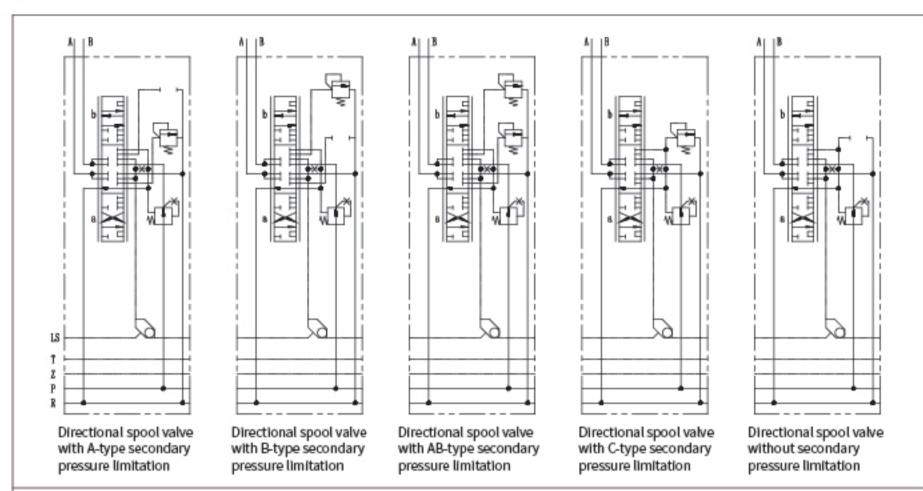
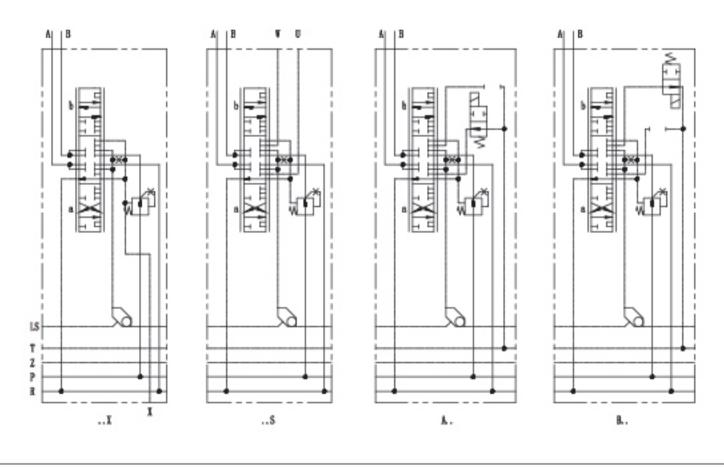


Table 10 Functional Cut-off and Pressure Extraction

Code	Description
N	Without functional cut-off and pressure extraction
х	C-type pressure limitation, with pressure extraction port X, thread size of oil port is G1/8 respectively
s	AB-type pressure limitation, with pressure extraction ports U & W, thread size of oil port is G1/8 respectively
S1	AB-type pressure limitation, with flange plate extraction ports U & W, thread size of oil port is G1/8 respectively
Α	Electrical cut-off on the side of actuator port A
В	Electrical cut-off on the side of actuator port B





Code	Operation manner
Α	Manual operation with long handle
A1	Manual operation without handle
A2	Manual operation with short handle
М	Manual friction positioning with long handle
E	Electric proportional control
EA	Combination of manual operation and electric proportional control with long handle
EA1	Combination of manual operation and electric proportional control without handle
EA2	Combination of manual operation and electric proportional control with short handle
D	Electric switching value control
DA	Combination of manual operation and electric switching value control with long handle
DA1	Combination of manual operation and electric switching value control without handle
DA2	Combination of manual operation and electric switching value control with short handle
Н	Pure hydraulic control, two pilot oil ports are in the same direction with oil ports A and B
_	Pure hydraulic control, two pilot oil ports are perpendicular to oil ports A and B and outward
F	respectively
	Hydraulic control and manual operation, two pilot oil ports are in the same direction with oil
HA	ports A and B with long handle
ГА	Hydraulic control and manual operation, two pilot oil ports are perpendicular to oil ports A and
FA	and outward respectively with long handle
HA1	Hydraulic control and manual operation, two pilot oil ports are in the same direction with oil
ПАТ	ports A and B without handle
HA2	Hydraulic control and manual operation, two pilot oil ports are in the same direction with oil
HAZ	ports A and B with short handle
FA1	Hydraulic control and manual operation, two pilot oil ports are perpendicular to oil ports A and
171	B and outward respectively without handle
FA2	Hydraulic control and manual operation, two pilot oil ports are perpendicular to oil ports A and
	B and outward respectively with short handle
Р	Pneumatic control
PA	Combination of manual operation and pneumatic control with long handle
PA1	Combination of manual operation and pneumatic control without handle
PA2	Combination of manual operation and pneumatic control with short handle
D1 41	Combination of manual operation and pneumatic control without handle, pneumatic control
P1A1	port is in reverse direction with oil ports A & B
P1A2	Combination of manual operation and pneumatic control with short handle, pneumatic control
FIME	port is in reverse direction with oil ports A & B

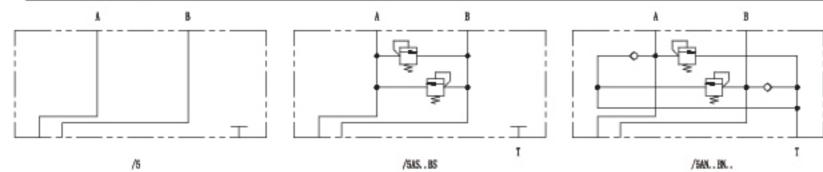
Name	Manual control		Electric hydraulic control		Hydraulic control		Pneumatic control		
	Spring restoration A(1,2)	Friction positioning M		Combination with manual control EA(1,2)	Pure hydraulic control HA,FA	Combination with manual control HA(1,2)FA(1,2)		Combination with pneum- atic control PA(1,2)	Combination with pneu- matic control PA(1,2)
Name and diagram code	MINITED AND THE PROPERTY OF TH		B A S	MAN TO THE PART OF	B A B A B A B A B A B A B A B A B A B A	B A B A B A B A B A B A B A B A B A B A	B A R	2 1 b B A A A A A	MATA THE PROPERTY OF THE PROPE
Operating parameter				Control current ratic I/I <sub>N</sub> :0.2-1;		Control the pressure to 5bar-18bar and maximum operating pressure to 50bar		the control to 7bar	

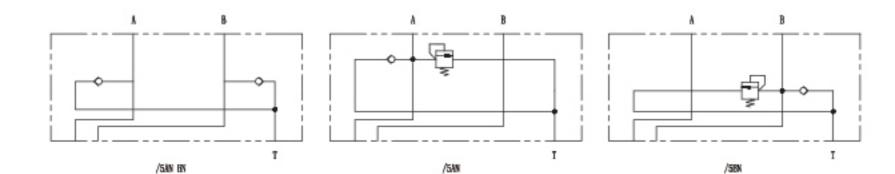
— Combined Type Size 5 — Combined Type Size 5

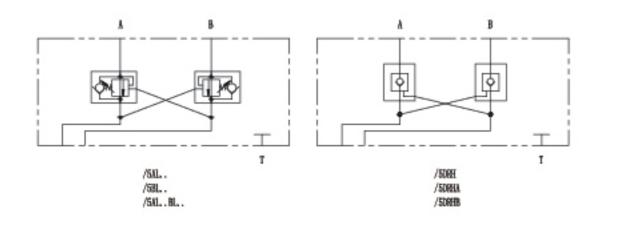


### Table 12 Ancillary Block

Code	Description
/5	Ancillary block without any additional function, thread sizes of oil ports shall be G1 respectively
/5ASBS	Pressure limiting safety valve is mounted at ports A & B, thread size of oil ports is G1, with limited pressure, unit: bar
/5AN BN	With ancillary block of oil supplementary valve at ports A & B
/5AN	With cushion valve at port A and ancillary block of oil supplementary valve at port B
/5BN	With cushion valve at port B and ancillary block of oil supplementary valve at port A
/5AL,/5BL	With ancillary block of balance valve at ports A & B, with the pressure indicated, unit: bar
/5DRH	With ancillary block of hydraulic control one-way valve at ports A & B
/5DRHA	With ancillary block of hydraulic control one-way valve at port A
/5DRHB	With ancillary block of hydraulic control one-way valve at port B







### 3.3 End plate

For example: HLPSL5 C 1 F/210-5-52L 80/40 B250 A /E- E1- G24 Table 13 Table 14



Table 13 End Plate

Code	Pilot oil return manner	Description
E1	Return to oil tank directly through leakage oil port T	Standard end plate, external discharge of pilot oil, connect- ing oil return control port T externally
E2	Return to oil tank directly through leakage oil port T	Similar to E1, with additional port Y so that it can connect with port LS of valve type HLPSV arranged separately
E3	Return to oil tank directly through leakage oil port T	External discharge of pilot oil, a 2-position 3-way magnetic ball valve is mounted; in case of power supply, the system pressure rises to the pressure of safety valve
E4	Internal oil return or oil return through external connection	Similar to E1, internal discharge of pilot oil, port T can be sealed
E5	Internal oil return or oil return through external connection	Similar to E2, internal discharge of pilot oil, port T can be sealed
E6	Internal oil return	Similar to E3, internal discharge of pilot oil, port T can be sealed
E18	Return to oil tank directly through leakage oil port T	Similar to E2/E5, with additional load oil port Y, pressure oil port P and oil return port R
E19	Internal oil return or oil return through external connection	Similar to E1/E4, with additional load oil port Y, pressure oil port P and oil return port R
E20	Internal oil return or oil return through external connection	Similar to E2/E5, with additional load oil port Y, pressure oil port P and oil return port R

Note: the internal control oil return channel can only be used in the system with oil return pressure of less than 10bar.

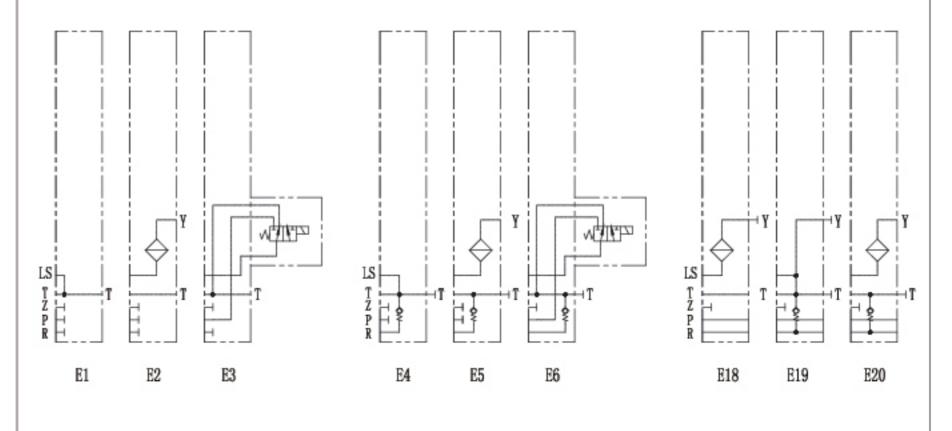


Table 14 Control Voltage and the Length of Anti-explosion Type Cable

G12	12VDC	The data of operation manner E is th
G24	24VDC	same with those in Section 4.3. As t
G24Ex-3m	24VDC, anti-explosion type, the guide line is 3m long	valve type HLPSL(V)-F or -D, the pum can be unloaded at any time.

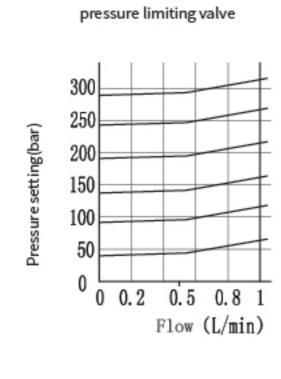
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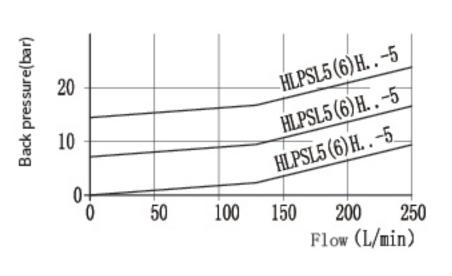
Parameters				
Overview and hydraulic pressure				
Model code	HLPSL and HLPSV			
Structure category	most, the material shall be	l spool valve, 12 e steel	directio	onal spool valves can be combined at
Fixed mode	threaded connection: M8			
Installation position	any			
Port	P= pressure oil inlet M= pressure gauge port R= oil return port Z= pilot pressure port (20 or 40bar) A, B= actuator port T= pilot control oil return port U, W, X= load pressure signal outlet Y= load pressure oil inlet (end plates E2, E5, E18 & E20) LS, DW= HLPSV-type load output port (note: not the pressure oil inlet)			
Port size	P, R, A, B= according to me M, LS, Z, T, Y, DW=G1/4 (co U, W, X =G1/8 (conforming	onforming to DIN	ISO 22	-
Surface treatment	all surfaces shall go through anti-corrosion surface phosphating.			e phosphating.
Quality (weight)	connection block		End	plate
	HLPSV5,6= about 8.1KG HLPSL5,6= about 8.1KG With magnetic ball valve, of D/F + 0.5	E3, E		2, E4, E5= about 2.5KG 6= about 3.2KG
	Directional spool valve Operation type	Standard type	e	Additional function C, ABS
	A, E, H, F	about 7.5KG		about 7.6KG
	EA	about 7.8KG		about 7.9KG
	HA, FA	about 7.6KG		about7.7KG
	HEA	about 8.2KG		about 8.3KG
	Transition block	Ancillary blo	ck	
	ZPL53 = about 2.6KG	/5		= about 1.6KG
	ZPL55 = about 2.6KG ZPL55/9 = about 1.0KG	/5ASBS		= about 3.1KG
	,	/5AN, /4BN		= about 3.1KG
		/5ALBL		= about 3.3KG
		/5AL,/3BL		= about 2.5KG
		/5DRH		= about 3.4KG
		/5DRHA, /5DR	HB	= about 2.5KG



Pressure medium	conform to ISO VG10-68 of DIN51519 Viscosity scope: about 4-1500mm2/s The best working range: about 10-500 mm2/s When the operating temperature doesn't exceed +70°C, HEPG-type synthetic medium (poly-alkyl ethylene glycol) and HEES-type synthetic ester can also be used, however HETG medium (colza oil) isn't applicable.
Temperature	environment temperature: about -40+80°C (note: it shall be about -40+40°C for anti-explosion type structural style) Oil temperature: about -25+80°C, pay attention to the viscosity scope (note: it shall be about -25+70°C for anti-explosion type structural style) In case the operating temperature is higher than at least 20K during subsequent work, the starting temperature is allowed to be decreased to -40°C (note: pay attention to the viscosity scope during the starting)
ggested pollution degree	ISO4406 18/14, NAS7~8 class
Working pressure	Pmax.=420bar, the maximum pressure of ports P, P1, A, B, LS, M & Y and slide valve actuate shall be reduced slightly, and the reduced value shall be equal to internal control pressure drop of fixed differential valve of the valve type HLPSL (please refer to "Load Pressure Flo Curve" on the next page) or internal control pressure drop at the pump flow regulating valve Oil return port R (50bar): the oil can return to the oil tank without pressure through separate pipeline from port T. However, in case the oil return pressure is relatively high, it's suggested to use end plate type E1, E2 and E3 with additional leakage port. The pressure at port Z is about 20 or 40bar (depending on the code in Table 3) (outlet) and 40bar (inlet).
Control oil way	internal control oil way, through filtration of disk filter, can prevent the fault caused by pollutio
Flow	maximum flow of the actuator shall be 16-160 (210) l/min
4.2 Characterist	ic curve



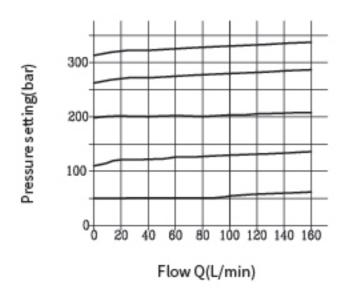
HLPSL: Median Cycle Pressure



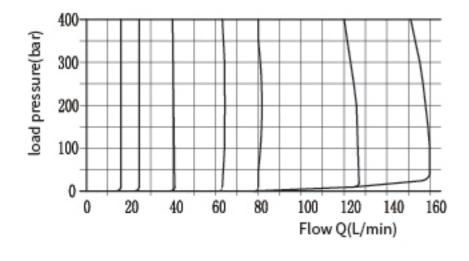
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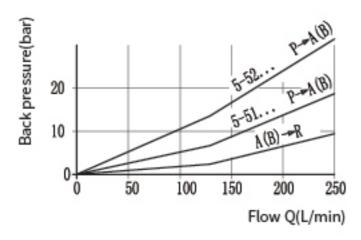
### 4.2.2 Directional spool valve



### Fixed differential reducing valve

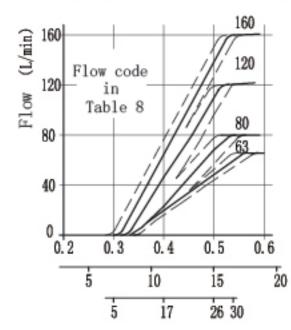


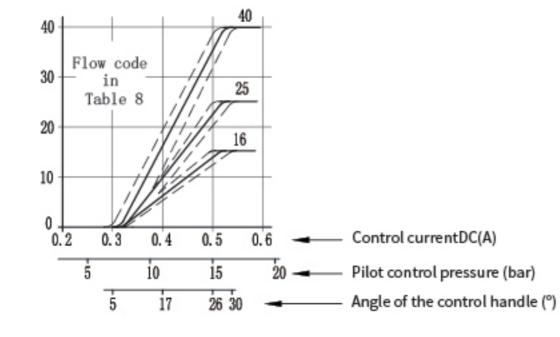
### full open back pressure



### Spool valve in&out flow curve

Input current-flow curve Actuator flow control curve (an example of directional spool valve with inlet fixed differential reducing valve)





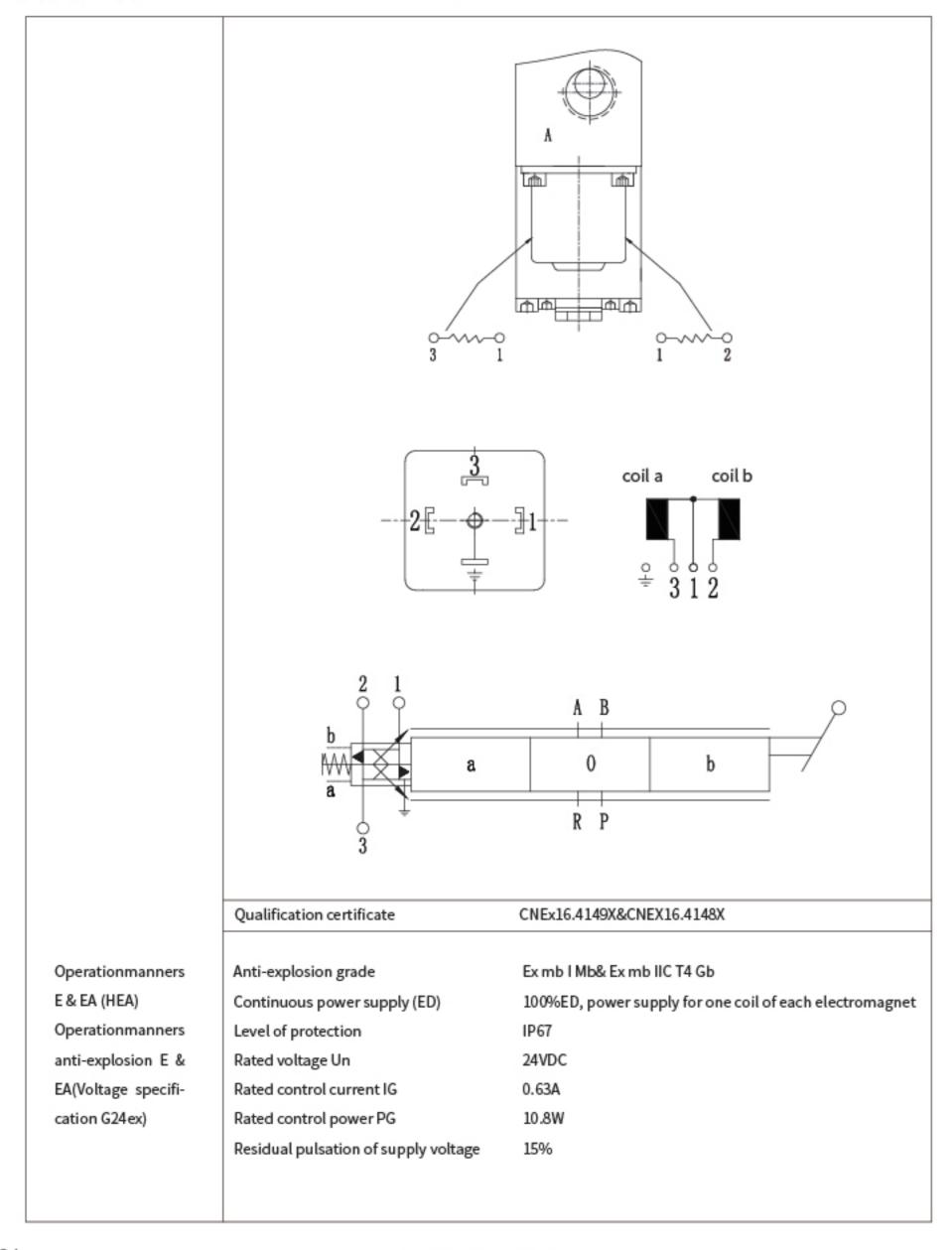


		_		HIDRAGEIC
4 3. Operating parameters				
4.3.1Operating torque	Neutral position	Terminal position	1	
Type A	about 3.0Nm	about 7.5Nm		
Туре НА	about 5.0Nm	about 16.5Nm		
Type EA	about 3.0Nm	about 12.0Nm		
4.3.2 Operation manner				
Operation manner M	Friction positioning	type, the valve cor	e can be positioned in termi	nal position
Operation manners E & EA		ication and mainte	ing iron cavity connecting ex nance are unnecessary for th	kternal seal and oil return ne linking iron, and hydraulic
	Date describes a library		24/06	121/00
	Rated voltage Un Coil resistance R20		24VDC	12VDC 6.7Ω
	Rated control curre	n+ IC	27.2Ω 0.63A	1.26A
			10.8W	10.6W
	Rated control powe Switching energy W		≤0.3Ws	≤0.3Ws
	Relatively continuo		100%	100%
	Level of protection		IP65	10070
	Flutter frequency	(assembled)	4070Hz (55Hz is the be	st)
	Flutter amplitude		20%≤Ad≤35%	
	Wiring of electric ap	pliance	DIN 43650A	
	Wiring diagram	,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,	coil a	coil b
		Cur	rent-valve core curve	
		100%		
		oftity of		
		25% Oben duar		
		0	0.2 0.5 0.8 1	
	60		Control current I/I <sub>N</sub>	out 60mm2/s
	60mm2/s	THE OIL VISCOSITY OF	uring the measurement is ab	out outilitiz/s.

— Combined Type Size 5 ——— Combined Type Size 5







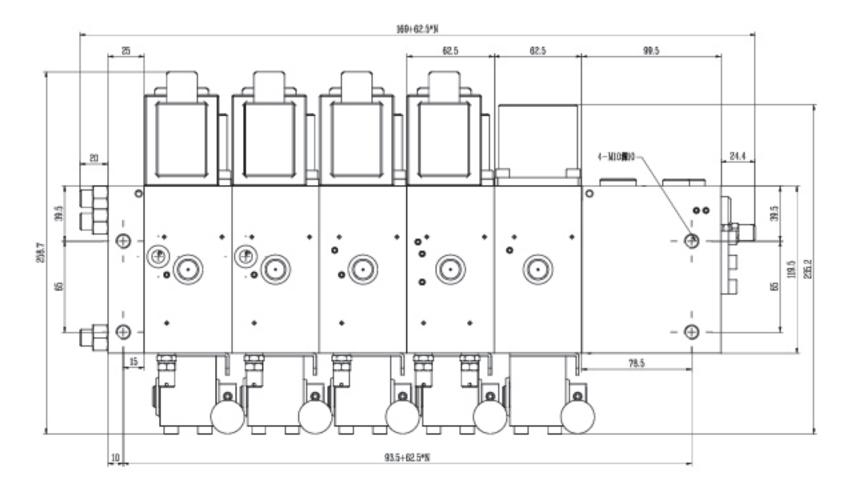
	Service conditions  Maximum environment temperature 40°C  Maximum medium temperature 70°C  Each electromagnet shall be protected by fuse  Surface treatment The enclosure shall be galvanized, and the coil and wiring cavity shall be sealed by rolling.  Note: prevent direct radiation of sunshine.  Cross section of the cable 4x1mm²  Cable length 3m/5m  Please refer to "Operation manners E & EA" (standard type) for the wiring diagram.
Operation manner H, F, HA, FA	the control pressure is about 5bar (at the beginning of the travel) -18bar (in terminal position); maximum allowable pressure is 50bar; remote control pipeline of control ports 1 & 2 shall be external connection; control oil shall be provided through proportional pilot valve.
4.4 Functional cut-off control	
Switching-type electromagnet with manual emergency operation	
Rated voltage Un	24VDC 12VDC
Coil resistance (R20)	34.8Ω 8.7Ω
Cold state current I20	0.68A 1.38A
Current gain 170	0.48A 0.97A
Power under normal temperature	16.6W 16.6W
Pn=Un x I20	
Level of protection (assembled)	IP 65
Wiring mode	DIN 43650A
Cut-off energy consumption WA	≤0.3Ws
Wiring diagram	coil b coil a
	coil a coil b $\frac{3}{2}$ $\frac{2}{2}$ $\frac{3}{2}$ $\frac{3}{2}$

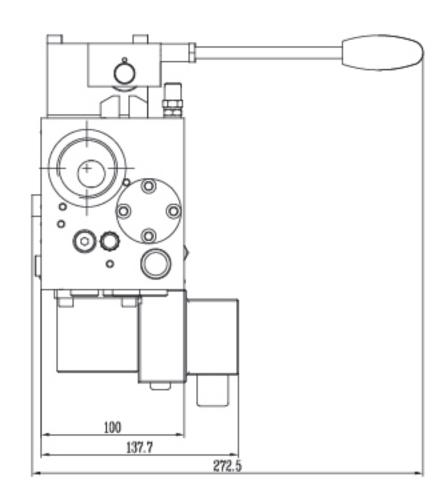
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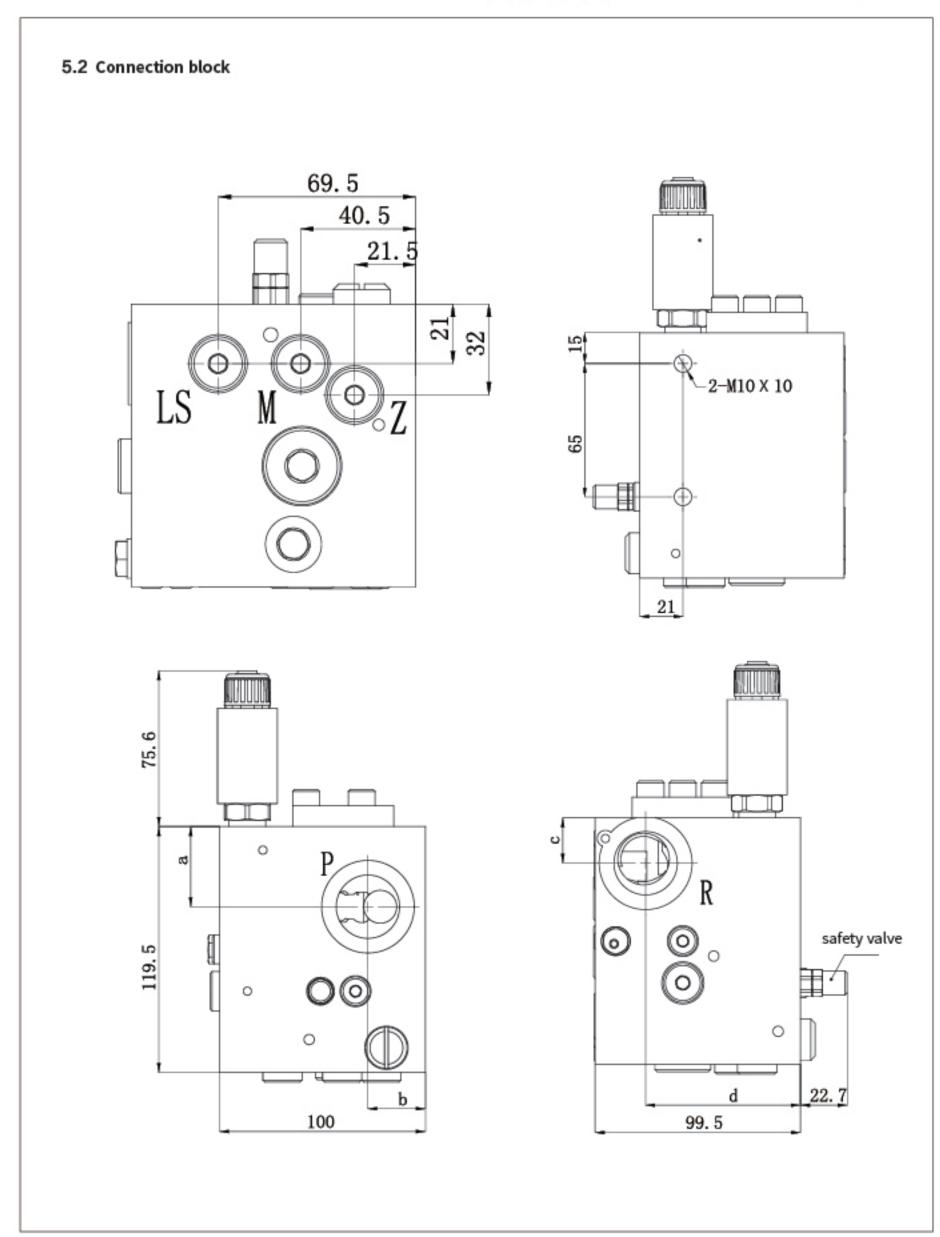
### V. Outline Dimension

### 5.1. Overall dimension





Note: mount the thread M8; "N" in the installation dimensions 170+62.5.5\*N and 93.5+62.5\*N refers to the number of ways of the directional spool valve. Users can confirm the installation dimension and outline dimension according to the actually required number of ways of the directional spool valve.





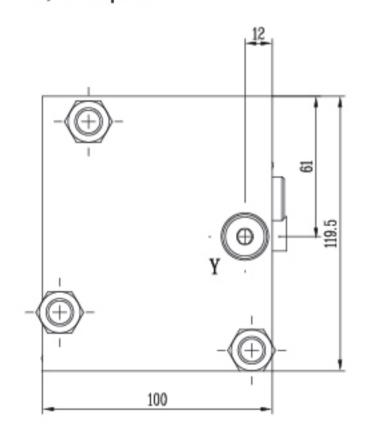
	HLPSL(V)5	HLPSL(V)6
а	35	35
b	68	68
с	28	28
d	73	73

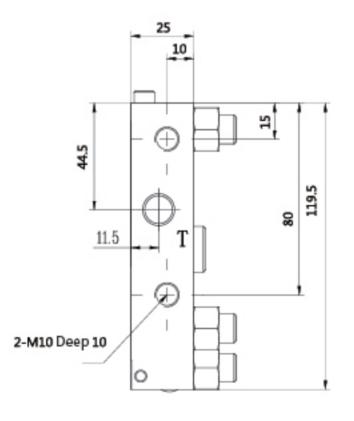
	P and R	LS, Z and M
HLPSL5/HLPSV5	G1	G1/4
HLPSL6/HLPSV6	G1 1/4	G1/4

Note: mount the thread M10; when the connection block HLPSL5(6) or HLPSV5(6) connects with the directional spool valve with auxiliary functions, ZPL55/9 transition plate shall be mounted between the connection block and directional spool valve according to actual demand. Otherwise pipe connector can't be mounted on oil return port

### 5.3 End plate

### 5.3.1 E1, E4 End plate





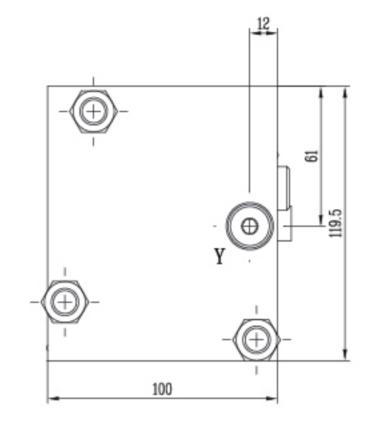
Oil port standard

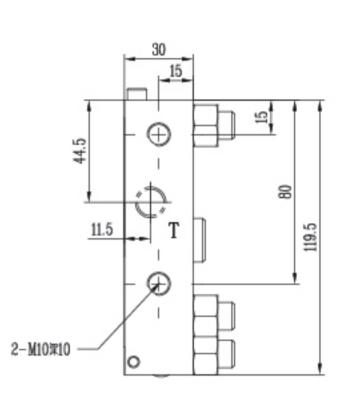
Please refer to DIN ISO 228/1 (BSPP)

T & Y =G1/4



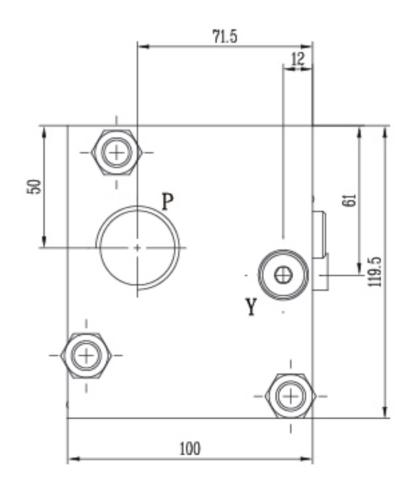
### 5.3.2 E2, E5 End plate

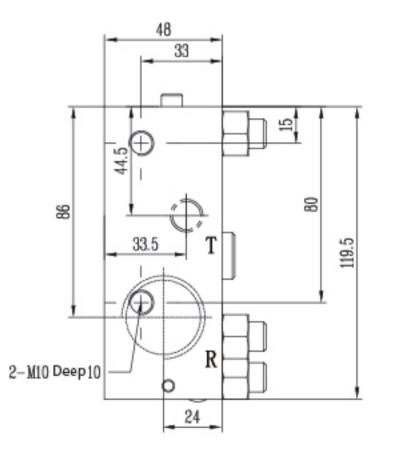




Oil port standard Please refer to DIN ISO 228/1 (BSPP) T & Y = G1/4

### 5.3.3 E18, E19, E20 End plate

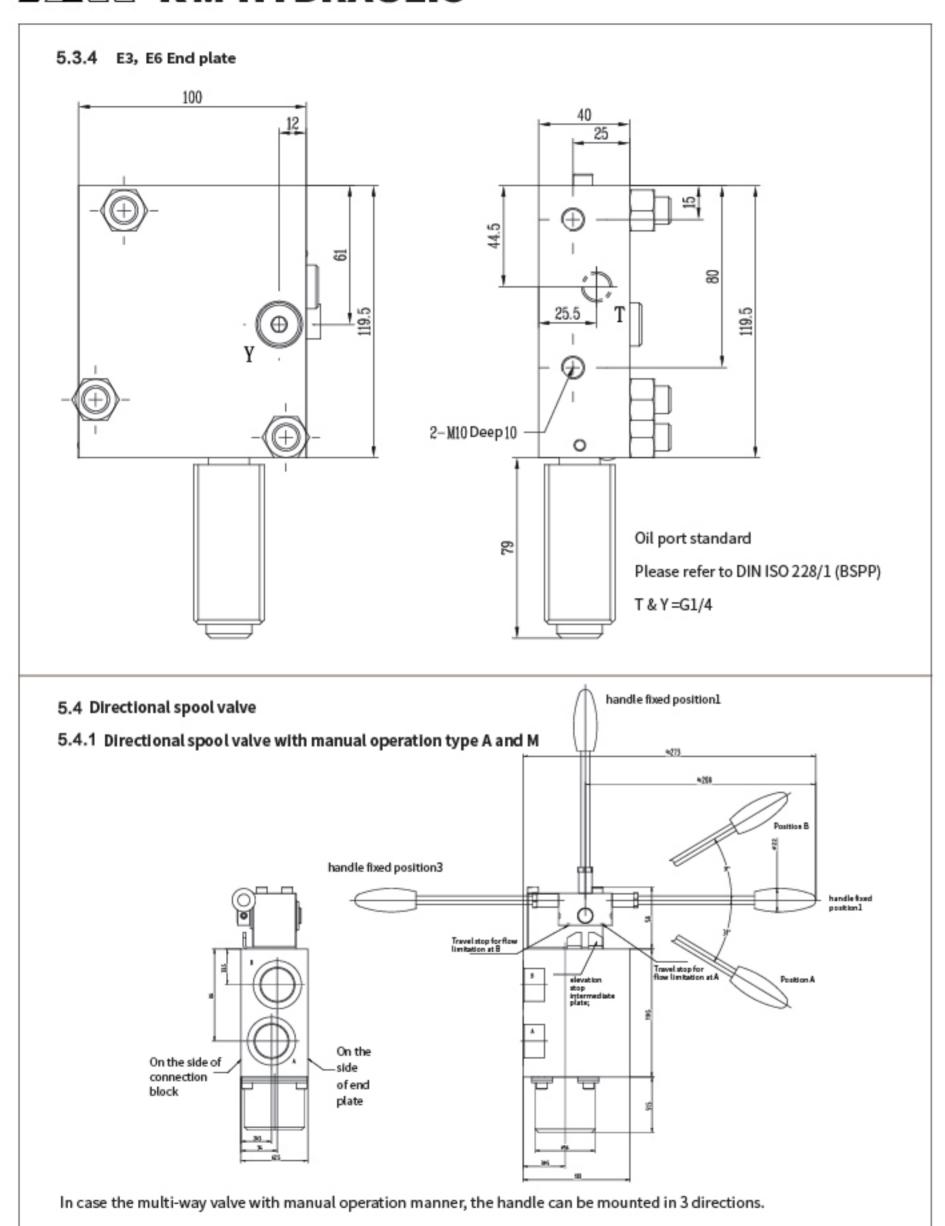


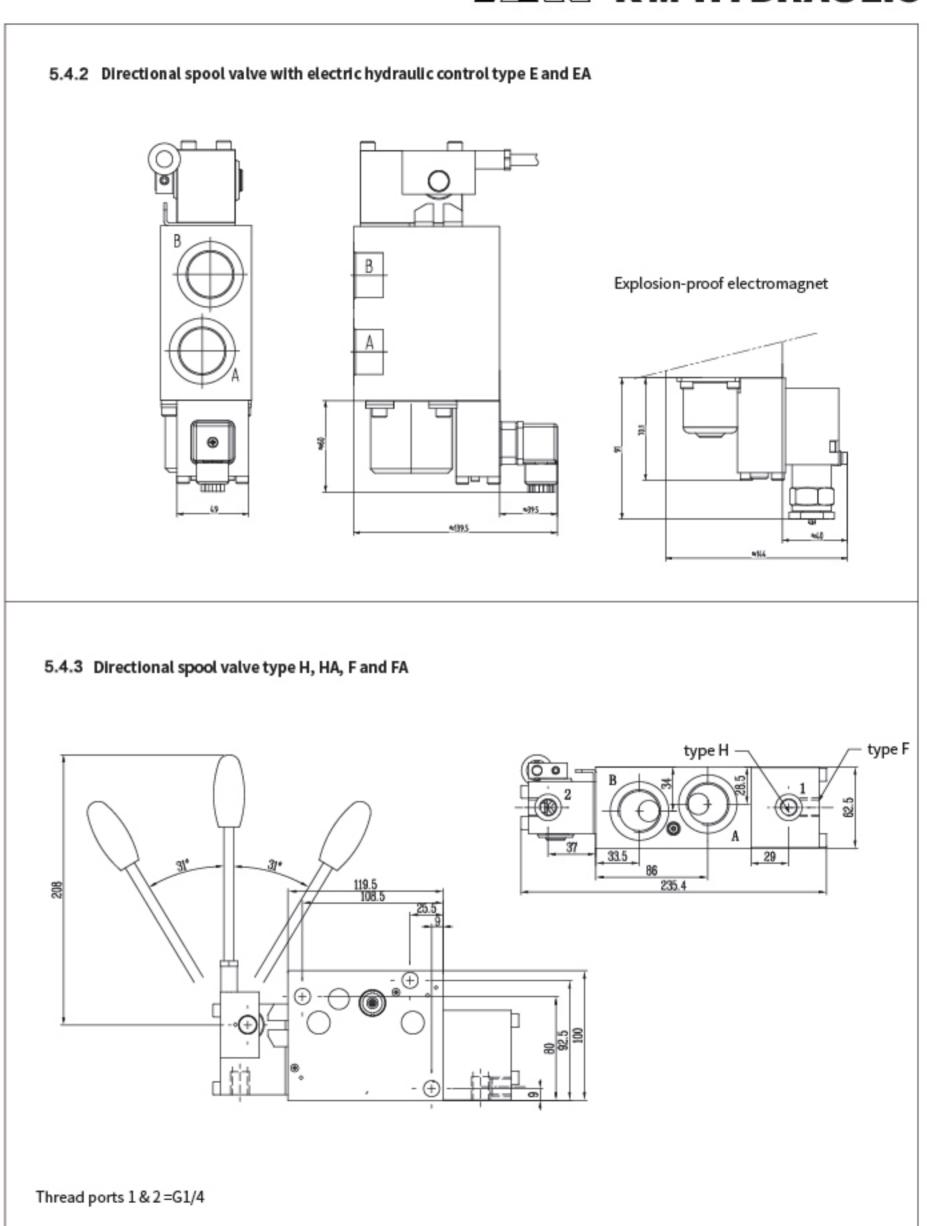


Oil port standard Please refer to DIN ISO 228/1 (BSPP) T & Y = G1/4 P & R = G1





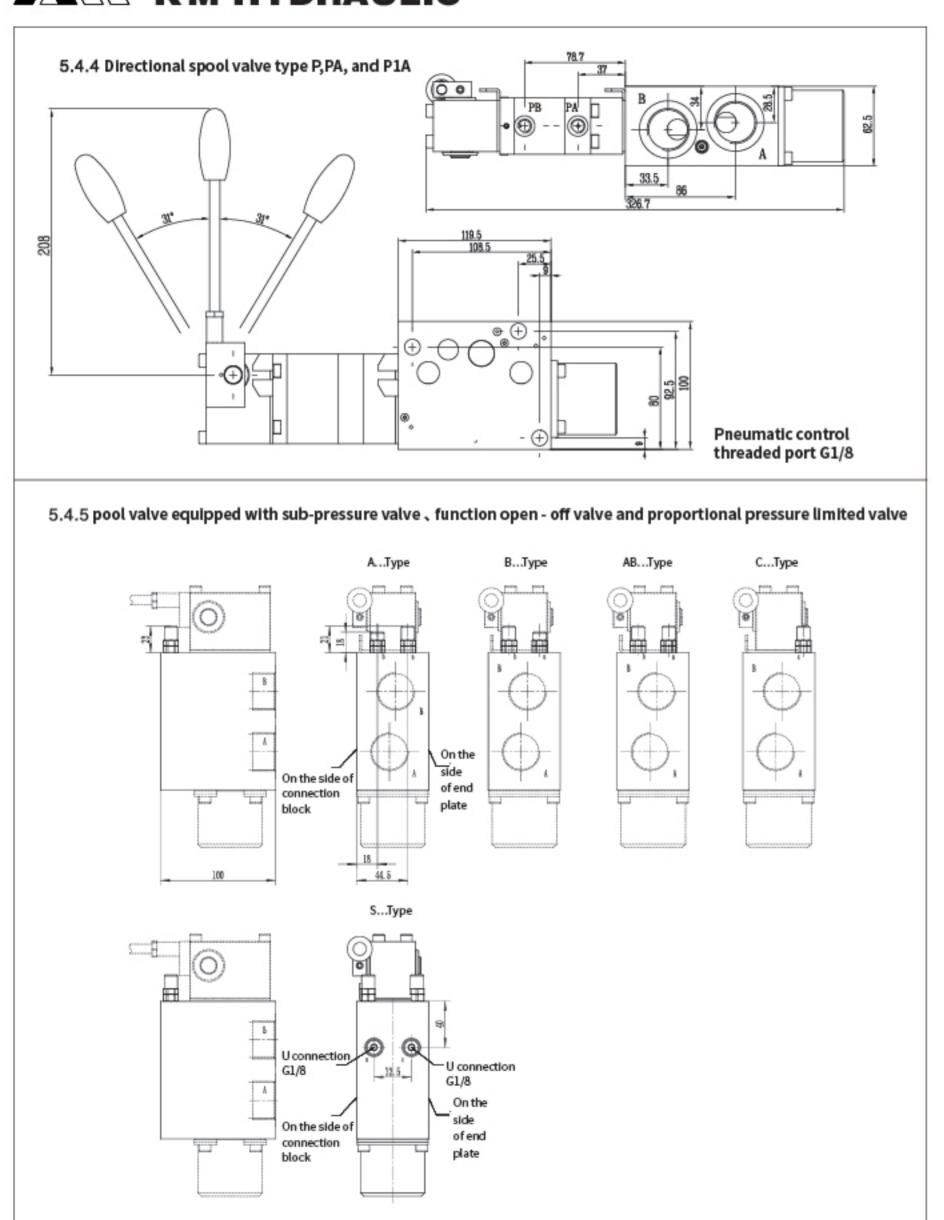


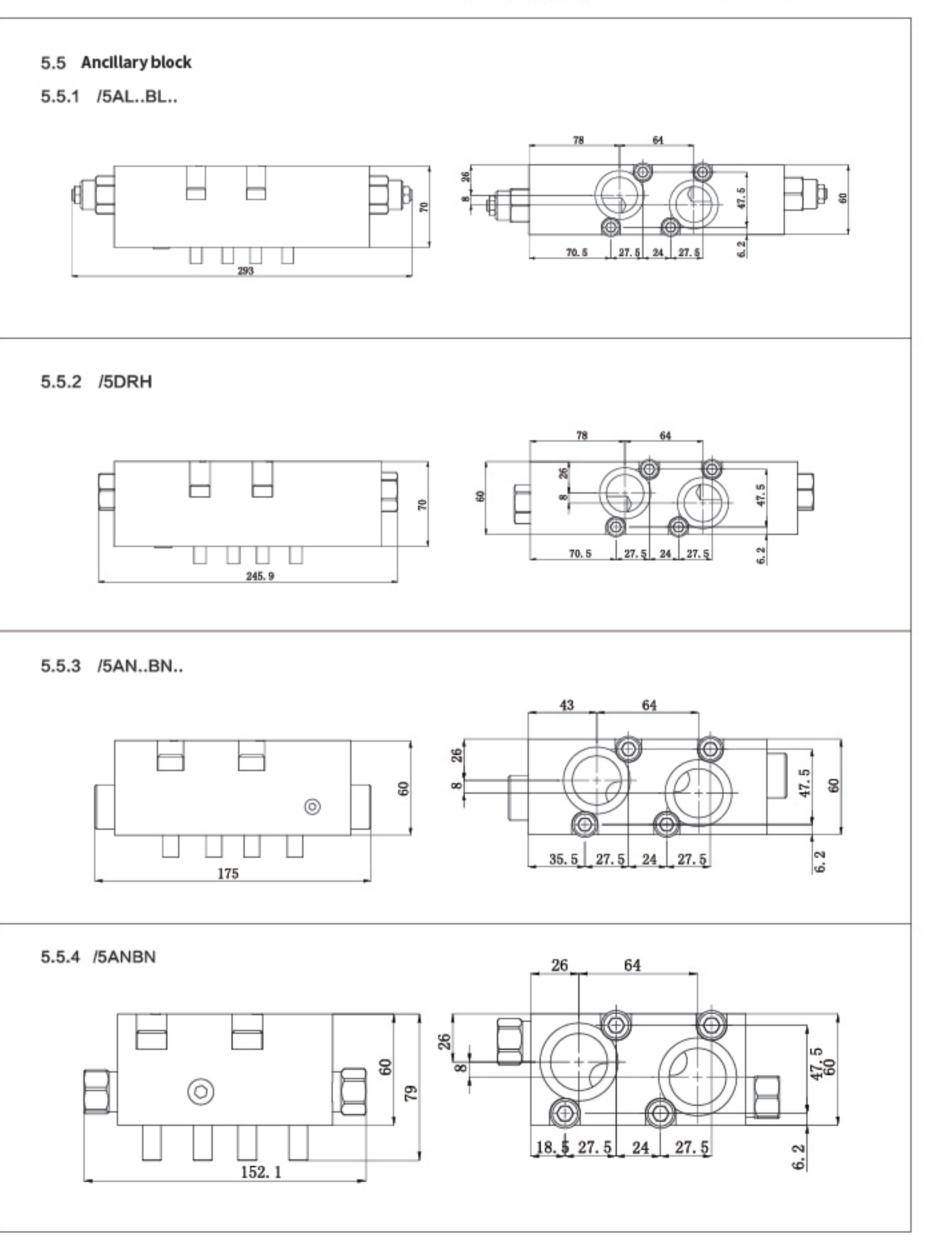


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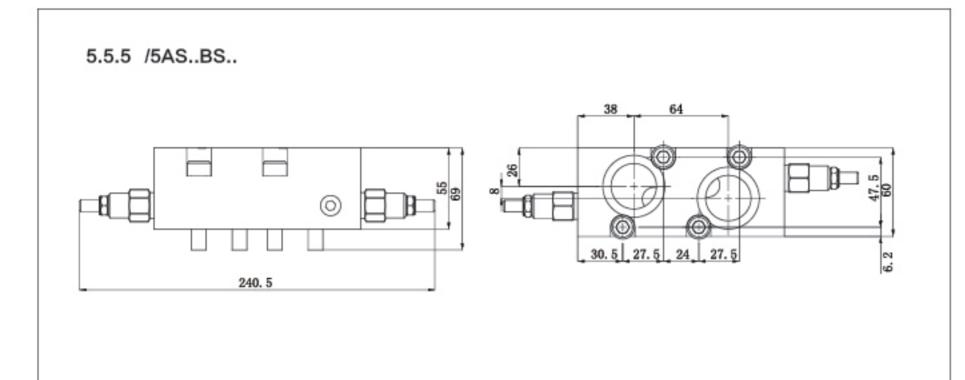






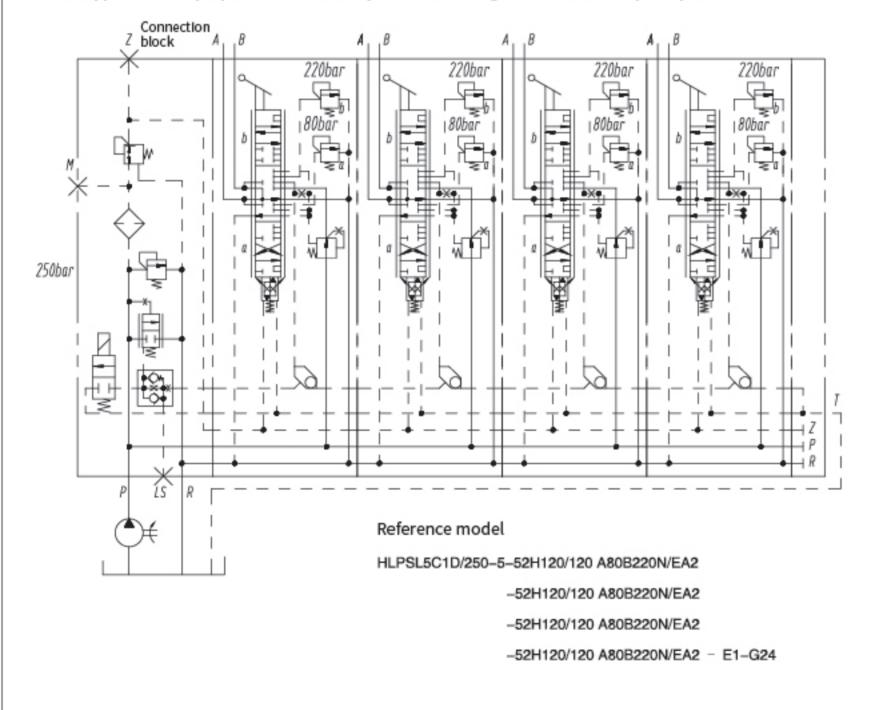


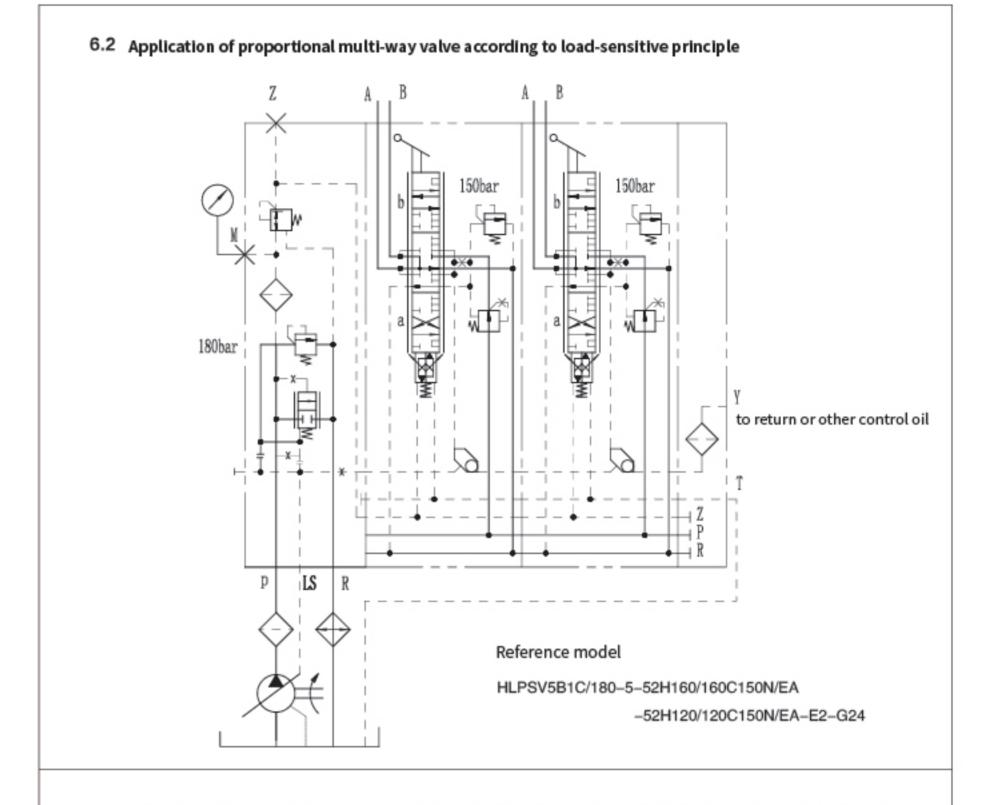




## VI. Application Example

6.1 Application of proportional multi-way valve according to load-sensitive principle





### 6.3 Application of proportional multi-way valve according to load-sensitive principle inMunicipal vehicles

Reference model

HLPSL5C1C/180-5-52L160/160NN/EA2

-E18-G24

HLPSV5C1C/180-5-A2H160/160NN/EA2/5AN

-A2H160/160NN/EA2/5AN

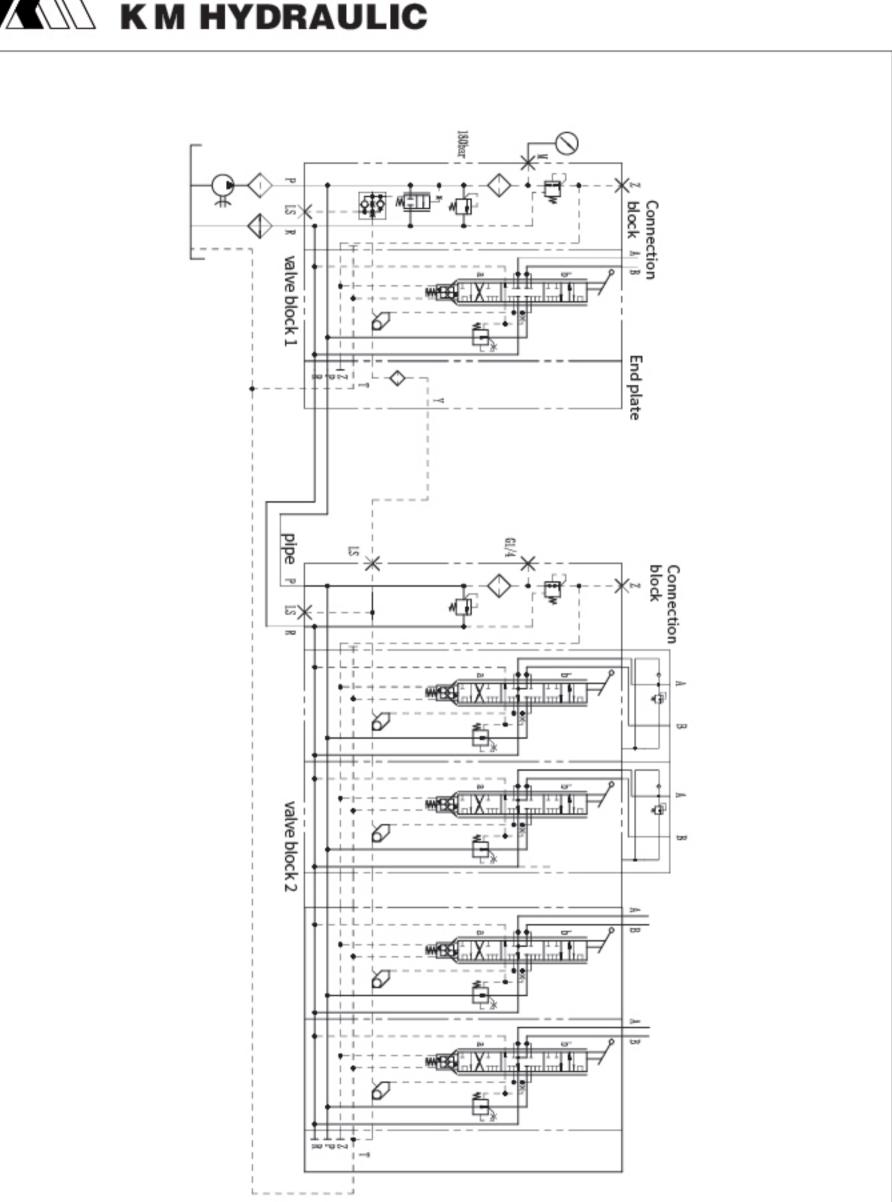
-ZPL5/3

-32H40/40NN/EA2

-32H40/40NN/EA2

-E1-G24







### VII. Other Characteristics

#### 7.1 Model selection and design description

#### 7.1.1 Modification for the use in special service conditions or with special requirements

Oceanic climate environment

In case of severe marine climate, all moving spare parts by manual operation manner shall go through sufficient anti-corrosion treatment. In this way, the hinge pin in handle operating rod bearing shall be made by stainless steel. All other spare parts shall either go through gas nitriding anti-corrosion treatment, or be made by stainless steel. Pressure impact in oil return way

Since extremely high pressure impact (>150bar) will be generated in the loop when the actuator is connected or broken, and it will cause slight leakage of the spring shield, enhanced-type spring shield can be used to prevent such phenomenon.

Note: the allowable oil return pressure shall be less than 50bar. As to relatively high oil return pressure, the reliability for operating the electromagnet can't be guaranteed.

#### 7.1.2 Use of variable displacement pump

In case of using the load-sensitive control and variable displacement pump together, LS signal oil way of pump's pressure-flow controller (load-sensitive regulator) shall be unloading in idling condition (when the actuator doesn't work) to reduce the circulation loss. Such constraint is realized through proportional directional spool valve. In case of no such pressure relief, the pump will still work with all surplus flow and under the pressure set by pressure regulator safety valve in non-reversing position.

Since some directional spool valves don't have such constraint, there shall be an internal bypass hole or throttle valve between the LS signal inlet and pressure release oil outlet on the pressure-flow controller of some varieties.

When using proportional directional spool valve type HLPSV, it's unnecessary to use abovementioned controller, and otherwise it will cause excessive discharge of control oil and result in functional fault. Due to functional reasons, the flow of control oil is limited to 2L/min specially (actuator runs at low speed). Note: it's required to block the possible bypass hole in pressure-flow controller.

#### 7.1.3 Combination with balance valve

In case the three control elements (three-way flow regulating valve in the pump or connection block, two-way flow regulating valve in the directional spool valve, and balance valve) are connected in series, vibration may occur for the control system due to external load change and resonance.

#### 7.1.3 Combination with over 12 directional spool valves

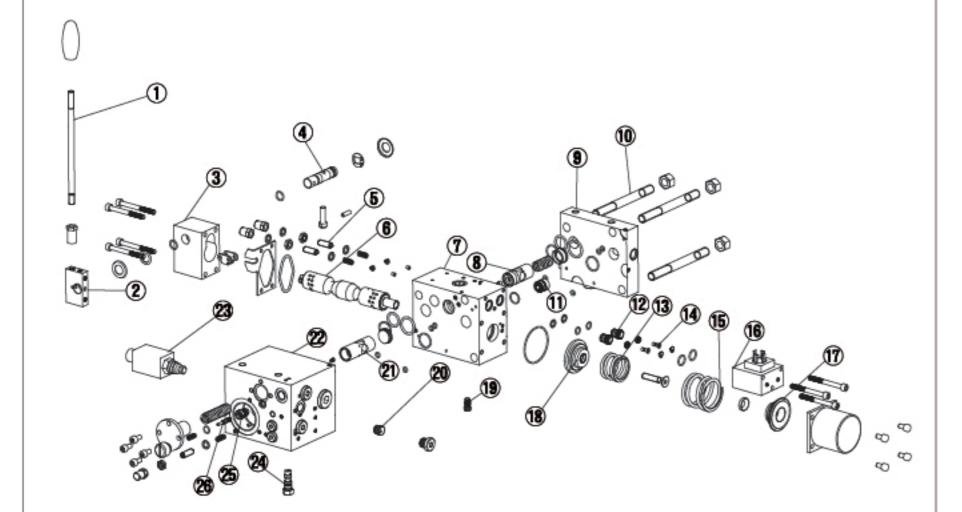
12 directional spool valves can be connected at most in the sequence of LS signal oil way no matter whether these directional spool valves are arranged in a valve bank or several valve banks. Such limitation is because the flow provided is limited (only can guarantee operation at low speed). In case over 12 directional spool valves are connected into several directional spool valve banks through LS signal respectively, chain oil way comprised of external shuttle valve shall be used.

73 — — — Combined Type Size 5 — — 74



# KM HYDRAULIC

### VIII. Explosive Drawing



- Handle lever
- 2 Limit stop
- (3) Handle seat
- 4 Shaft
- Secondary pressure limiting valve
- 6 Main valve core
- ⑦ Directional spool valve
- (8) Fixed differential reducing valve
- (9) End plate
- 10 Connecting rod
- 11 Pressure extraction plug
- Valve bush of proportional reducing valve So Valve bush of three-way reducing valve
- (13) Inner spring

- Valve core of proportional reducing valve
- 16 Outer spring
- 16 Electromagnet
- 17 Spring collar A
- 18 Spring collar B
- (19) Shuttle valve
- 20 Filter
- Fixed differential overflow valve
- Connection block
- Electromagnetic unloading valve
- Throttling damper
- Valve core of three-way reducing valve

### IX. Matters Needing Attention for Use

- 1、It's required to design and install a high-pressure pipeline filter with the precision of 5μ between the outlet of hydraulic pump and port P of proportional multi-way valve according to load-sensitive principle, because the proportional multi-way valve according to load-sensitive principle is assembled on a forging valve block through plug-in mounting of several hydraulic control elements. Since the system structure is complex with multiple ways and high coordinating precision of slide valve, the proportional multi-way valve according to load-sensitive principle has relatively high requirement on the cleanliness of hydraulic medium. To realize reliable operation of proportional multi-way valve according to load-sensitive principle, pollution control shall be carried out for hydraulic medium.
  - Circulating washing shall be carried out for hydraulic pipeline generally. If necessary, use a set of ordinary multi-way valve connection system to carry out working cycle for the whole hydraulic system to realize complete cleaning of the system and remove early failure of the system. Then install the proportional multi-way valve according to load-sensitive principle officially and carry out system debugging and ex-factory acceptance.
- 2. As to end plate E1, E2, E3 and E18, pilot oil return port T on the end plate shall be connected to oil return tank directly without back pressure and can't be combined with oil return way!
- As to end plate E4, E5, E6, E19 and E20, it's suggested to connect the pilot oil return port T on the end plate to oil return tank directly without back pressure. In this way, the working performance can be improved and service life can be prolonged.
- 4. When connecting & mounting the pipeline of proportional multi-way valve according to load-sensitive principle, oil port R on connection block of the multi-way valve is the general oil return inlet and shall be connected to the oil return tank without any mistake! In case oil port R on the connection block is connected wrongly, which causes high pressure or unopened stop valve mounted on the oil return pipeline, oil leakage and even explosion of proportional multi-way valve according to load-sensitive principle will be caused!
- 5. As to the multi-way valve according to load-sensitive principle with electric proportional control or electric switching value control, the two coils of double-end proportion electromagnet can't be connected with power supply for control simultaneously and the two coil windings shall control two directions of multi-way directional spool valve
- 6. As to the multi-way valve according to load-sensitive principle with electric proportional control or electric switching value control, it's required to check the parameters of control voltage and current at first, and then carry out control according to such parameters of multi-way valve to prevent any damage of double-end proportion electromagnet or electrical control equipment.
- 7. When the proportional multi-way valve according to load-sensitive principle is mounted on engineering machinery vehicle for use, rain-proof dust cover shall be designed & mounted for protection to prevent the multi-way valve from being exposed in rainy & sandy environment!

### X. Treatment of Simple Faults

1. The whole system has no pressure: at first, make sure that the liquid level of oil tank is 150mm higher than oil inlet and the rotating direction of hydraulic pump is correct; then check the correctness of connection from pump outlet to multi-way valve and check if the high-pressure pipeline filter is blocked by any contaminant; besides, check if additional overflow valve (safety valve) is in zero-pressure state, except for the multi-way valve; as to proportional multi-way valve according to load-sensitive principle (type HLPSV) with variable displacement pump system according to load-sensitive principle, it's also required to check if the oil way between oil port LS on connection block of the multi-way valve and port X on hydraulic pump is unblocked; check if the safety valve on connection block of the multi-way valve is in zero-pressure state; check if electromagnetic unloading valve on connection block of the multi-way valve powers on according to the requirement; in case the fault still can't be removed upon abovementioned inspections are completed, check and clean the throttling orifice one-way valve (throttling plug) on connection block of the multi-way valve; in case the fault still can't be removed, it's required to deliver the system back to Jiangsu KM Hydraulic Control System Co., Ltd. for repair.

Combined Type Size 5 



- One way of the multi-way valve has no pressure or output pressure is lower than the set pressure value: as to the directional spool valve with secondary pressure limitation, check if the secondary pressure limiting valve is loose at first; then check & clean the shuttle valve (between oil ports A and B) of the multi-way valve, check if the shuttle valve is polluted and if the steel ball is stuck. In case the fault can't be removed, maybe the throttling orifice of pressure extraction plug is blocked and it's required to disassemble the multi-way valve for cleaning.
- 3. One end (port A or port B) of one way of the multi-way valve has no pressure or output pressure is lower than the set pressure value: at first, check if the corresponding secondary pressure limiting valve is loose (what's next to the end plate is the secondary pressure limiting valve of port A). In case the fault can't be removed, maybe the throttling orifice of pressure extraction plug is blocked and it's required to disassemble the multi-way valve for cleaning.
- 4. Electric proportional control of multi-way is insensitive, namely, the valve will not work with small control signal and carry out snap action with large signal, which is because the pilot proportional reducing valve of directional spool valve of the way is polluted and the movement of valve core of the pilot proportional reducing valve is insensitive. Steps for removing the fault: a) clean the dusts on related proportion electromagnet of multi-way valve and on surrounding outer surface; b) disassemble related proportion electromagnet and pay attention to not losing internal valve core of proportional reducing valve and conical spring; c) inspection: press the valve core of two proportional reducing valves by index finger and middle finger respectively, and check if the upper end face of valve core can move flexibly without any clamping stagnation from the position parallel with the plane of valve body to the area 1-2mm lower than the plane; d) disassemble the valve core of proportional reducing valve and conical spring for cleaning and mount them in corresponding valve seat again; e) re-inspection: press the valve core of two proportional reducing valves by index finger and middle finger respectively, and check if the upper end face of valve core can move flexibly without any clamping stagnation from the position parallel with the plane of valve body to the area 1-2mm lower than the plane; f) check if the push rod of proportion electromagnet can move flexibly; g) mount the proportion electromagnet on the valve block of directional spool valve; h) carry out experimental verification again to remove the fault.
- 5. Upon the multi-way valve according to load-sensitive principle with electric proportional control or electric switching value control is started up without power supply, the handle of multi-way valve moves on its own, namely, there is output of the multi-way valve, oil cylinder or motor begins to work, which is because the valve core of pilot proportional reducing valve of directional spool valve of the way is stuck. Please refer to above item 4 for the method for fault removal.
- 6. As to the multi-way valve according to load-sensitive principle with electric proportional control or electric switching value control, manual control is normal, while electronic control doesn't work, namely, the directional spool valve of each way doesn't work in case of electric operation. At first, check if the control electric signal is normal. In case the electric signal is normal, maybe the valve core of 3-way reducing valve on the connection block is stuck. Unscrew the valve seat (slotted) of 3-way reducing valve and clean the valve core. In this way, the fault can be removed.
- Replacement of two rotary seal rings of the handle. Since the two seal rings are dynamic seal and oil leakage will occur after they' re used for 2 years generally, it's required to replace them. Steps for replacement: a) disassemble spring shield of the directional spool valve; b) disassemble inner & outer springs and spring seat (or friction positioning mechanism); c) disassemble the handle seat; d) pull the main valve core out of the valve body; e) take the main valve core off; f) take the stop block of handle, check ring and its dust ring off; g) push the spindle respectively and replace rotary seal ring; h) recover the assembly according to reverse sequence of the disassembling order; l) matters needing attention for disassembling & assembling: check the seal ring of each part; prevent the entry of any contaminant; as to main valve core with the flow equal to or less than 16L/min, pay attention to the mounting direction of main valve opening, and the **pressure extraction groove on the step in the middle of main valve core shall face to the direction of end plate!**

77 — Combined Type Size 5 — Combined Type Size 5



Reply with your comments will be appreciate.

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